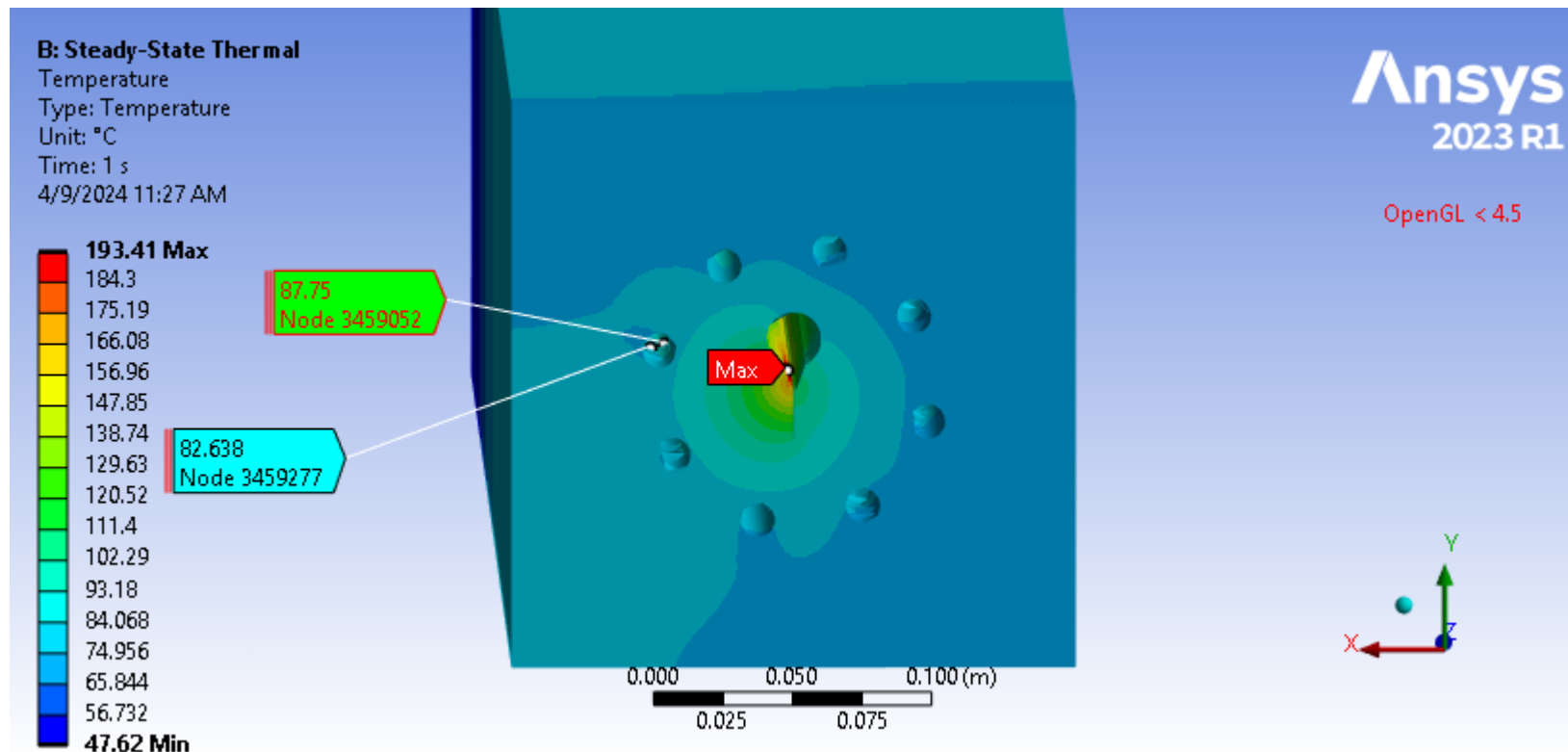
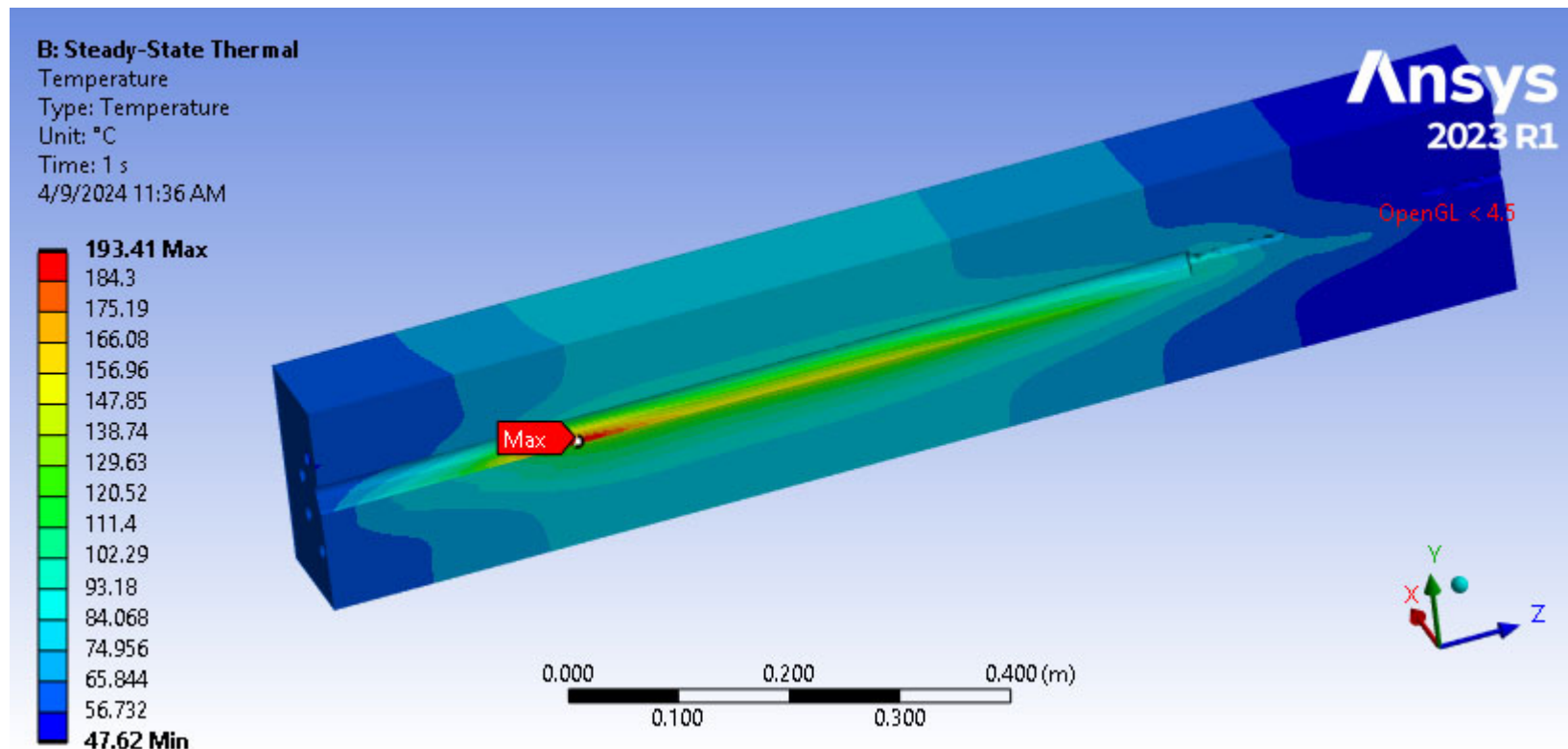


Pavel 78, .2mm mesh at bottom of keyhole, 40C water, 5000w/M²-K, ½” holes on 5cm from center. 51.14KW , 1mm X offset



Pavel 78, .2mm mesh at bottom of keyhole, 40C water, 5000w/M²-K, ½” holes on 5cm from center. 51.14KW , 1mm X offset



Max wall temp at water channel = 90C

$h = 5059 \text{ W/m}^2\text{-K}$

KLong CPS 52KW total, 4circuits LCW, max pump cap. 20 gpm

Units	Units	Units	Units	Units	Units	Units	Units	Units	Units	Units	Units	Units
d	12.7 mm	0.0416667 ft	0.0127 m	ID of tube								
L	10 m	10000 mm										
epsilon			0.000005 ft	e/d =	0.00012							
nu	0.00000929 ft ² /sec	at 37°C										
Coil Power	13 kW	13000 W										

Heat Exchange with water at 70 psi DP

T _{water} =	51.85336988 C	average	62.70674 m
N _{ud} =	62	From Oliver & Meyer pag	
K =	0.623 W/MK	From White pg 550	
Pr =	4.84	extrapolate from White appendix F	
f =	0.027407511	friction factor	
h =	Nud K/d	W/M ² K	
h =	5059.700713		
q =	hA(T _w -T _{wa})	=	mCpdeltaT =
A =	piDL	used for area inside cooling blocks, 2	
T _{wall} =	80.09755618 ave		90.95093 max

$$v = -2 \sqrt{\frac{2g\Delta P d}{0.433 L}} \log_{10} \left(\frac{\epsilon}{3.7d} + \frac{2.51}{\frac{d}{v} \sqrt{\frac{2g\Delta P d}{0.433 L}}} \right)$$

$$\frac{1}{\sqrt{f}} = -2 \log_{10} \left(\frac{\epsilon}{3.7d} + \frac{2.51}{\frac{d}{v} \sqrt{\frac{2g\Delta P d}{0.433 L}}} \right)$$

$$q \left(\frac{\text{gpm}}{\text{circuit}} \right) = \frac{v \pi d^2}{4}$$

$$= v \left(\frac{\text{ft}}{\text{sec}} \right) \frac{\pi d^2 (\text{ft}^2)}{4} \times \frac{\text{gal}}{0.1337 \text{ ft}^3} \times 60 \frac{\text{sec}}{\text{min}}$$

$$Re = \frac{vd}{\nu}$$

$$\Delta T = \frac{3.8P}{q}$$

DeltaP (psi)	$\sqrt{\frac{2g\Delta P d}{0.433 L}}$ (ft/sec)	(no units)	(no units)	f	v (ft/sec)	Re	q (gpm)	DT (deg.C)	V	DP	h
2	0.614632957	0.0009429	6.0510294	0.0273112	3.7191621	16680.849	2.275790863	21.70674			
3.2	0.777456028	0.0007523	6.247271	0.0256224	4.8569785	21784.08	2.9720316	16.621627			
8	1.229265914	0.0004877	6.6237167	0.0227927	8.1423091	36519.147	4.98235682	9.9149864	8.1423091	8	
20	1.94364007	0.0003204	6.9887206	0.0204741	13.583557	60923.741	8.311908658	5.9432799	13.583557	20	
40	2.748722147	0.000236	7.2540703	0.0190036	19.939424	89430.498	12.2011241	4.0488073	19.939424	40	
45	2.915460105	0.0002244	7.298013	0.0187755	21.277066	95429.968	13.01963995	3.7942678	21.277066	45	
50	3.073164786	0.0002145	7.3370058	0.0185764	22.547828	101129.48	13.7972315	3.5804284	22.547828	50	
55	3.223162419	0.0002061	7.3720129	0.0184004	23.761195	106571.56	14.5397025	3.3975936	23.761195	55	
60	3.366483352	0.0001987	7.4037437	0.018243	24.92458	111789.47	15.25158889	3.2390068			
65	3.503946966	0.0001921	7.4327351	0.018101	26.04391	116809.78	15.93651731	3.099799			
70	3.636217612	0.0001863	7.459403	0.0179718	27.124012	121654.16	16.5974426	2.9763622			
75	3.76384281	0.0001811	7.4840761	0.0178535	28.168886	126340.54	17.23681085	2.8659594			
80	3.887280139	0.0001764	7.5070189	0.0177445	29.181885	130883.95	17.85667477	2.7664725			
85	4.006916653	0.0001721	7.5284467	0.0176437	30.165858	135297.18	18.45877729	2.6762336			
90	4.12308322	0.0001682	7.5485376	0.0175499	31.123249	139591.18	19.04461364	2.5939093			
95	4.236065323	0.0001645	7.5674404	0.0174623	32.056172	143775.44	19.6154782	2.5184194			

2 para circuits | air-n2 Hx

Fig. 7. Heat transfer results for the fully developed smooth tube